

ME 321: FLUID MECHANICS-I

Dr. A.B.M. Toufique Hasan

Professor

Department of Mechanical Engineering

Bangladesh University of Engineering and Technology (BUET), Dhaka

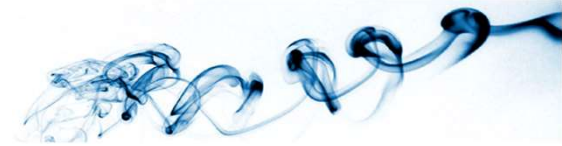
Lecture-10

01/03/2024

Measurements in Fluid mechanics

toufiquehasan.buet.ac.bd
toufiquehasan@me.buet.ac.bd



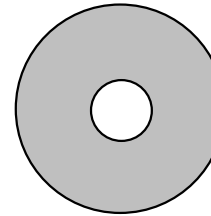
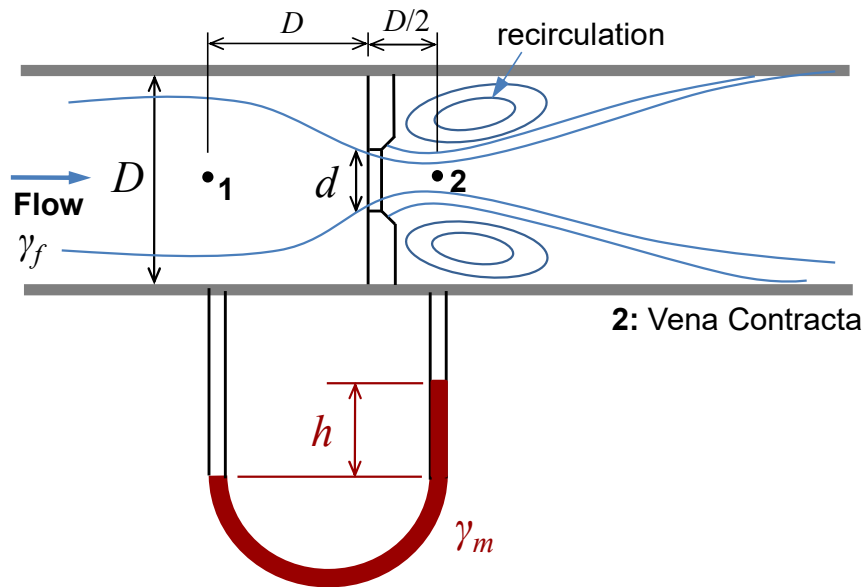


Measurement of

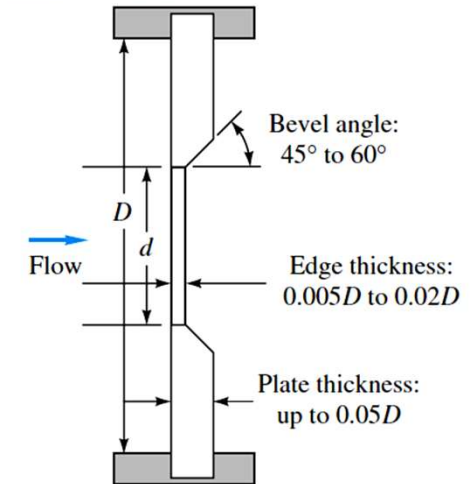
- flow rate (volume flow rate)
- flow velocity



Orifice meter



Orifice plate



Continuity equation: $Q = \frac{\pi}{4} D^2 V_1 = \frac{\pi}{4} d^2 V_2$

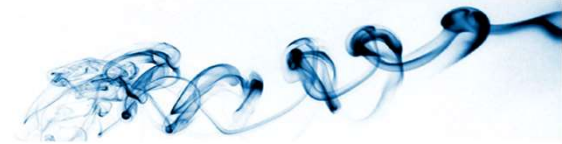
Bernoulli equation between point 1 and 2:

$$\frac{p_1}{\gamma_f} + \frac{V_1^2}{2g} + z_1 = \frac{p_2}{\gamma_f} + \frac{V_2^2}{2g} + z_2$$

$$\Rightarrow \frac{p_1}{\gamma_f} + \frac{V_1^2}{2g} = \frac{p_2}{\gamma_f} + \frac{V_2^2}{2g}$$



Orifice meter



$$\Rightarrow \frac{V_2^2 - V_1^2}{2g} = \frac{p_1 - p_2}{\gamma_f}$$

$$\Rightarrow V_2^2 - V_1^2 = \frac{2(p_1 - p_2)}{\rho_f}$$

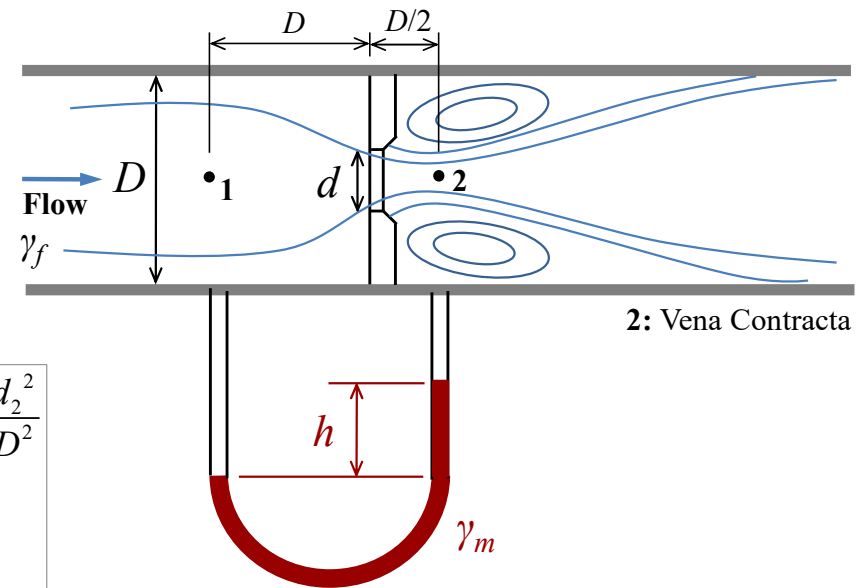
$$\Rightarrow V_2^2 \left(1 - \frac{V_1^2}{V_2^2}\right) = \frac{2(p_1 - p_2)}{\rho_f}$$

$$\Rightarrow V_2^2 \left(1 - \frac{d_2^4}{D^4}\right) = \frac{2(p_1 - p_2)}{\rho_f}$$

$$\Rightarrow V_2 = \sqrt{\frac{2(p_1 - p_2)}{\rho_f \left(1 - \frac{d_2^4}{D^4}\right)}}$$

$$Q = \frac{\pi}{4} D^2 V_1 = \frac{\pi}{4} d_2^2 V_2 \quad \therefore \frac{V_1}{V_2} = \frac{d_2^2}{D^2}$$

$$\therefore V_1 = \frac{Q}{\frac{\pi}{4} D^2} \quad \text{and} \quad V_2 = \frac{Q}{\frac{\pi}{4} d_2^2}$$

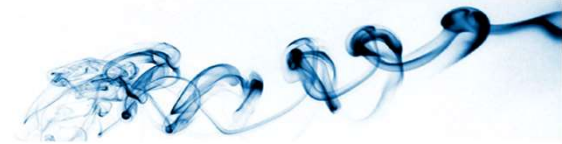


Then the theoretical volume flowrate could be expressed as:

$$Q_{theo.} = A_2 V_2 = \frac{A_2}{\sqrt{\left(1 - \frac{d_2^4}{D^4}\right)}} \sqrt{\frac{2(p_1 - p_2)}{\rho_f}} \quad (i)$$



Orifice meter



It is not convenient to use point 2 (vena contracta), instead orifice diameter, d is used which could be available from geometric configuration.

Define a coefficient, namely “coefficient of contraction, C_c ” as:

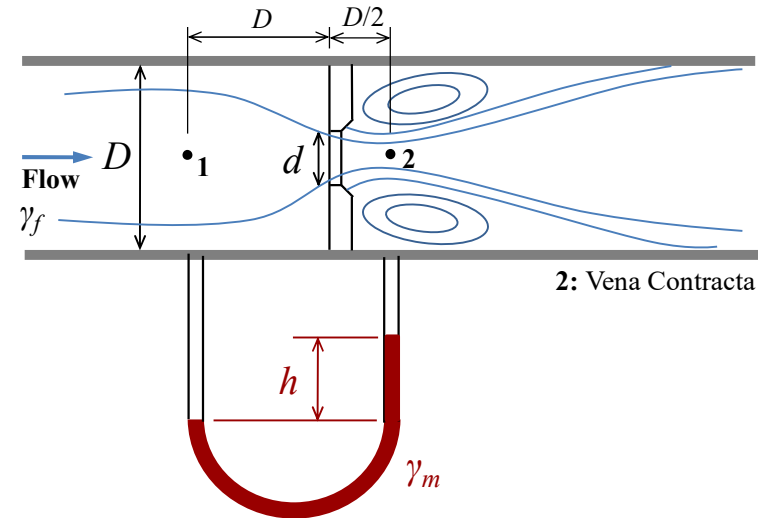
$$C_c = \frac{A_2}{A_0} = \frac{d_2^2}{d^2} \left(\equiv \frac{\text{area at vena contracta}}{\text{area at the orifice}} \right)$$

Then Eq. (i) comes as:

$$Q_{theo.} = \frac{C_c A_0}{\sqrt{\left(1 - \frac{C_c^2 d^4}{D^4}\right)}} \sqrt{\frac{2(p_1 - p_2)}{\rho_f}} \quad (ii)$$

However, assuming $C_c \approx 1$, then Eq. (ii) can be written as:

$$Q_{theo.} \approx \frac{A_0}{\sqrt{\left(1 - \frac{d^4}{D^4}\right)}} \sqrt{\frac{2(p_1 - p_2)}{\rho_f}} \quad (iii)$$



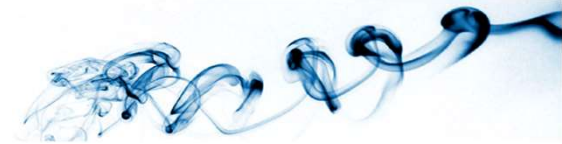
$$Q_{theo.} \approx \frac{A_0}{\sqrt{(1 - \beta^4)}} \sqrt{\frac{2(p_1 - p_2)}{\rho_f}} \quad (iv)$$

where $\beta = \frac{d}{D}$

The consideration of vena contracta (point 2) is accommodated by measuring the downstream pressure at point 2 (NOT at the orifice).



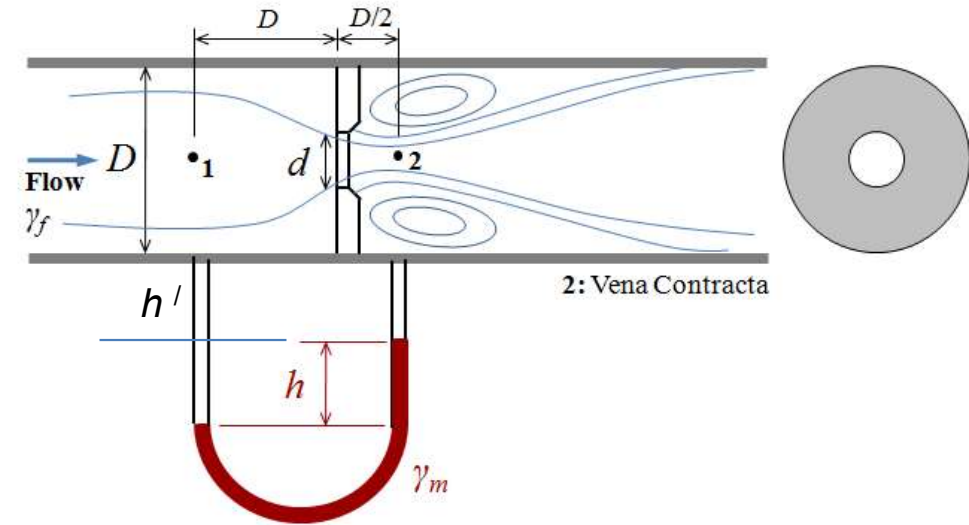
Orifice meter



Frictional (viscous) effect becomes very important while such obstruction meter is used in flow systems. To accommodate such effect, empirical **discharge coefficient, C_d** is defined as:

$$C_d = \frac{Q_{actual}}{Q_{theo.}}$$

$$\therefore Q_{actual} = C_d Q_{theo.} = \frac{C_d A_0}{\sqrt{(1-\beta^4)}} \sqrt{\frac{2(p_1 - p_2)}{\rho_f}} \quad (v)$$



Pressure differential can be measured using different approaches; for example using U-tube differential manometer as:

$$p_1 + (h' + h)\gamma_f = p_2 + h'\gamma_f + h\gamma_m$$

$$\Rightarrow p_1 + h\gamma_f = p_2 + h\gamma_m$$

$$\Rightarrow p_1 - p_2 = h(\gamma_m - \gamma_f)$$

$$\Rightarrow \frac{p_1 - p_2}{\rho_f} = h \left(\frac{\gamma_m - \gamma_f}{\rho_f} \right)$$

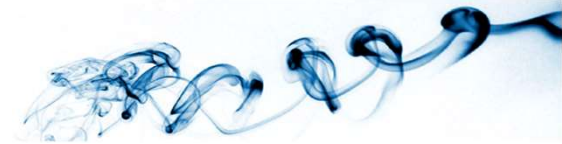
$$\Rightarrow \frac{p_1 - p_2}{\rho_f} = gh \left(\frac{\rho_m}{\rho_f} - 1 \right)$$

$$\Rightarrow \frac{p_1 - p_2}{\rho_f} = gh \left(\frac{S_m}{S_f} - 1 \right)$$

specific gravity (SG), S



Orifice meter



$$\therefore Q_{actual} = C_d Q_{theo.} = \frac{C_d A_0}{\sqrt{(1-\beta^4)}} \sqrt{\frac{2(p_1 - p_2)}{\rho_f}} \quad (v)$$

$$\therefore Q_{actual} = \frac{C_d A_0}{\sqrt{(1-\beta^4)}} \sqrt{2gh \left(\frac{S_m}{S_f} - 1 \right)}$$

$$\therefore Q_{actual} = \frac{C_d A_0}{\sqrt{(1-\beta^4)}} \sqrt{2gH}$$

where $H = h \left(\frac{S_m}{S_f} - 1 \right)$

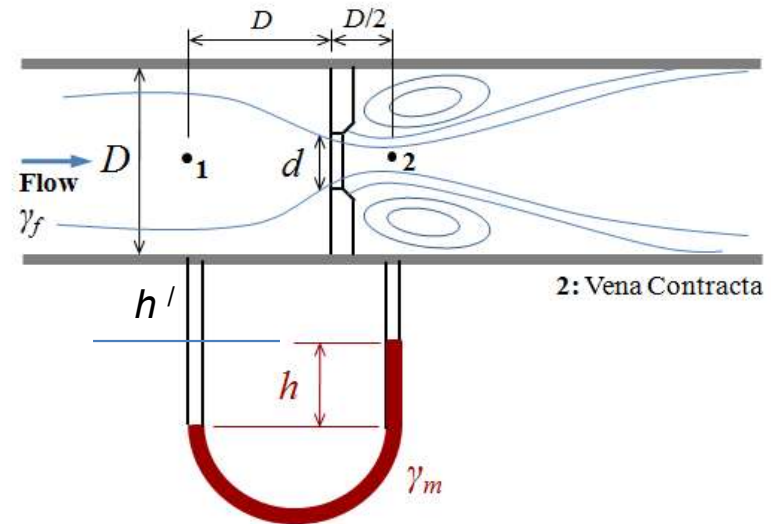
S_f is the specific gravity (SG) of the flowing fluid

S_m is the specific gravity (SG) of the manometric fluid

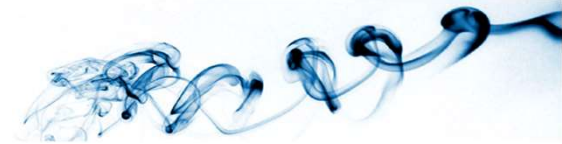
In short form:

$$Q_{actual} = K \sqrt{2gH} \quad ; \quad K = \frac{C_d A_0}{\sqrt{(1-\beta^4)}}$$

$$\text{or, } Q_{actual} = K_1 \sqrt{H} \quad ; \quad K_1 = \frac{C_d A_0}{\sqrt{(1-\beta^4)}} \sqrt{2g}$$



Orifice meter



ASME recommends the use of curve-fit formula for C_d developed by ISO according to:

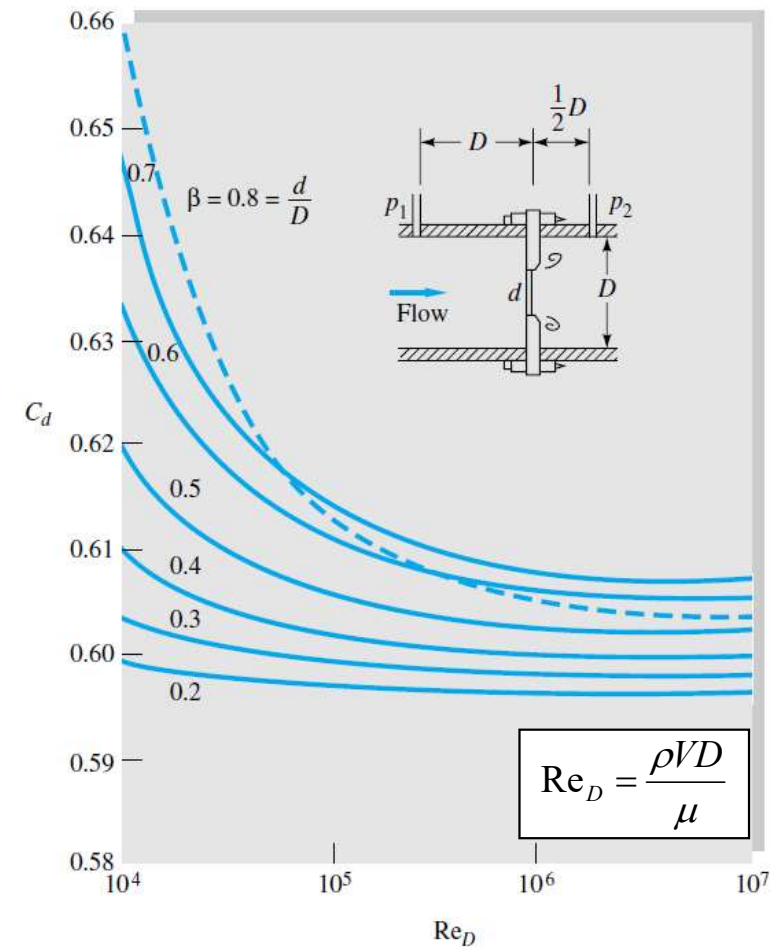
$$C_d = f(\text{Re}_D, \beta)$$

$$C_d = f(\beta) + 91.71\beta^{2.5} \text{Re}_D^{-0.75} + \frac{0.09\beta^4}{1-\beta^4} F_1 - 0.0337\beta^3 F_2$$

where:

$$f(\beta) = 0.5959 + 0.0312\beta^{2.1} - 0.184\beta^8$$

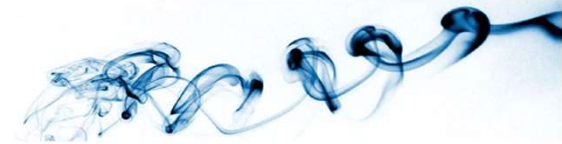
$$F_1 = 0.4333 \quad F_2 = 0.47 \quad (D : \frac{1}{2} D \text{ taps})$$



Discharge coefficient of an orifice in the range of Reynolds number 10^4 to 10^7



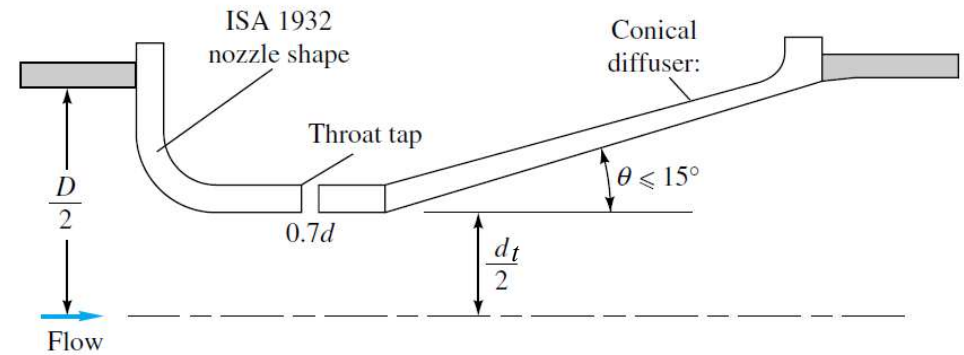
Venturi meter



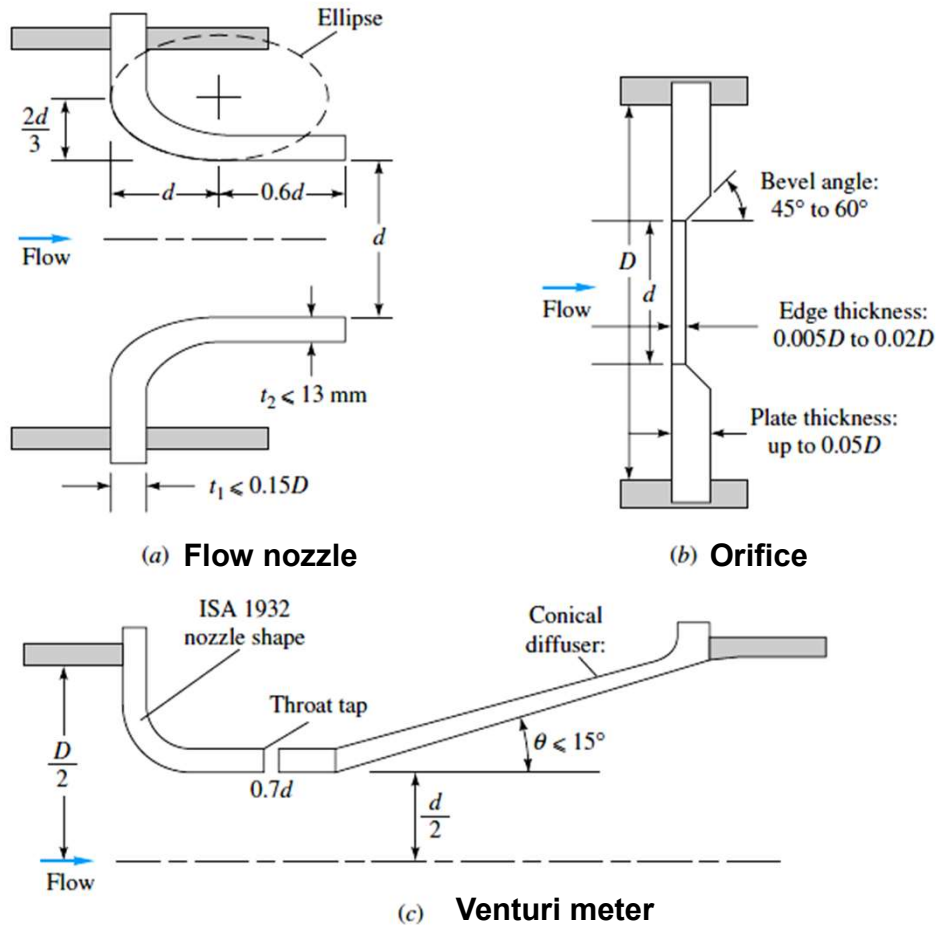
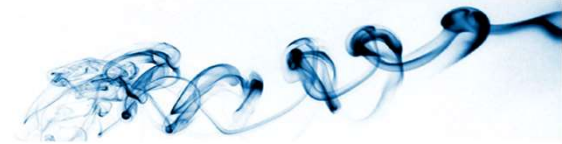
Modern venturi meter consists of an ISA 1932 nozzle entrance and a conical expansion of half angle no greater than 15 deg. Its discharge coefficient is given by the ISO correlation formula:

$$C_d \approx 0.9858 - 0.196\beta^{4.5}$$

$$\beta = \frac{d_t}{D}$$



Flow meters



Type of meter	Net head loss	Cost
Orifice	Large	Small
Nozzle	Medium	Medium
Venturi	Small	Large

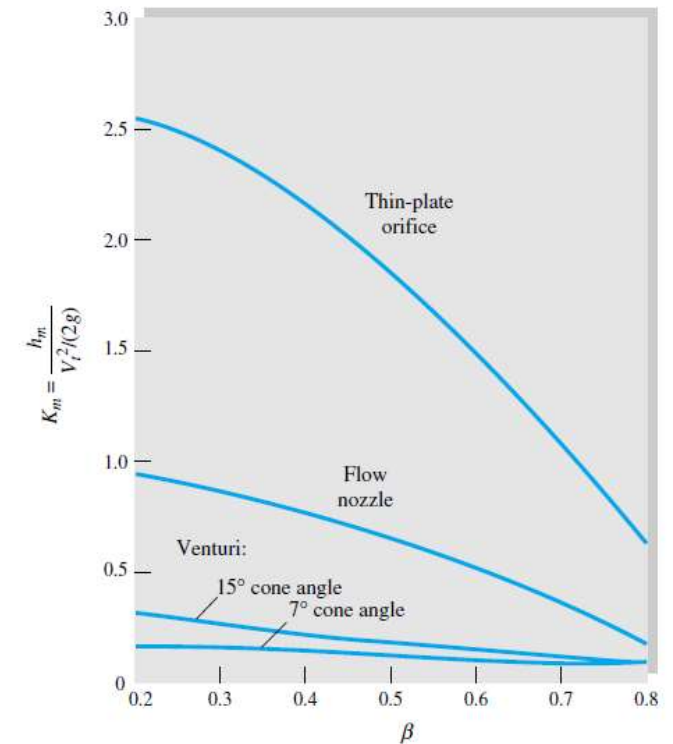
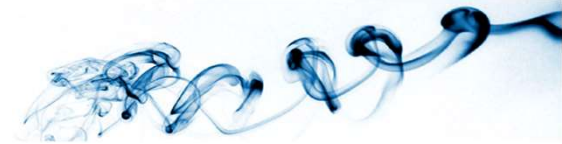


Fig. 6.43 Nonrecoverable head loss in Bernoulli obstruction meters. (Adapted from Ref. 30.)



Pitot tube



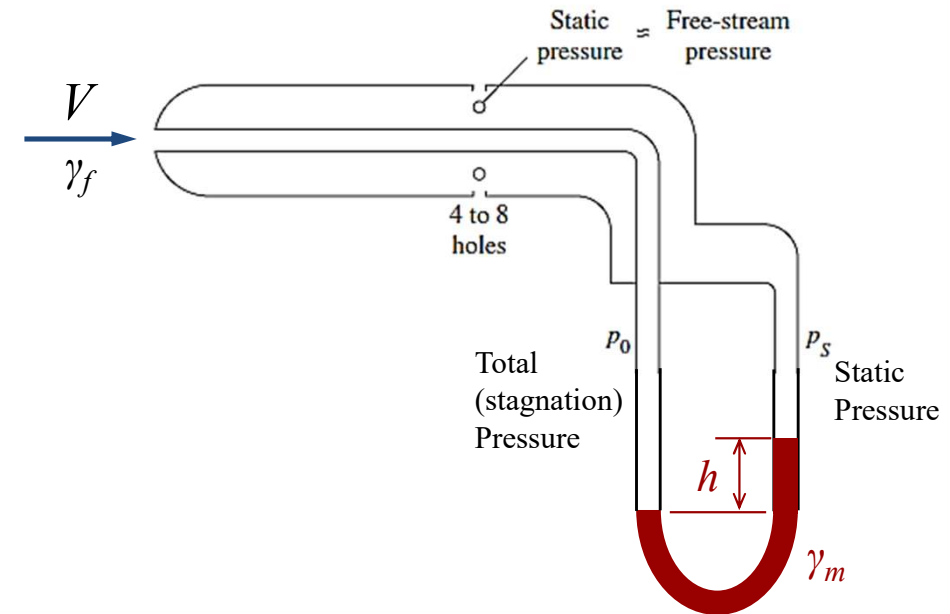
For local velocity measurement inside duct, wind tunnel etc.

Apply Bernoulli equation between stagnation point and static point:

$$\frac{p_0}{\gamma_f} + \frac{V_0^2}{2g} + z_0 = \frac{p_s}{\gamma_f} + \frac{V_s^2}{2g} + z_s$$

$$\Rightarrow \frac{p_0}{\gamma_f} = \frac{p_s}{\gamma_f} + \frac{V^2}{2g}$$

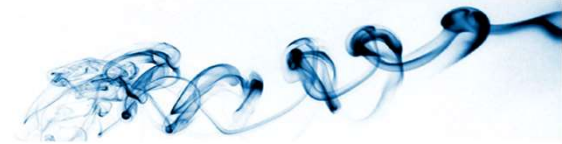
$$\Rightarrow V = \sqrt{\frac{2(p_0 - p_s)}{\rho_f}}$$



Pitot tube (pitot-static tube)



Pitot tube



Now from principle of manometry

$$p_0 + \gamma_f h_1 = p_s + \gamma_f (h_1 - h) + \gamma_m h$$

$$\Rightarrow p_0 = p_s - \gamma_f h + \gamma_m h$$

$$\Rightarrow p_0 - p_s = h(\gamma_m - \gamma_f)$$

$$\Rightarrow p_0 - p_s = \gamma_f h \left(\frac{\gamma_m}{\gamma_f} - 1 \right)$$

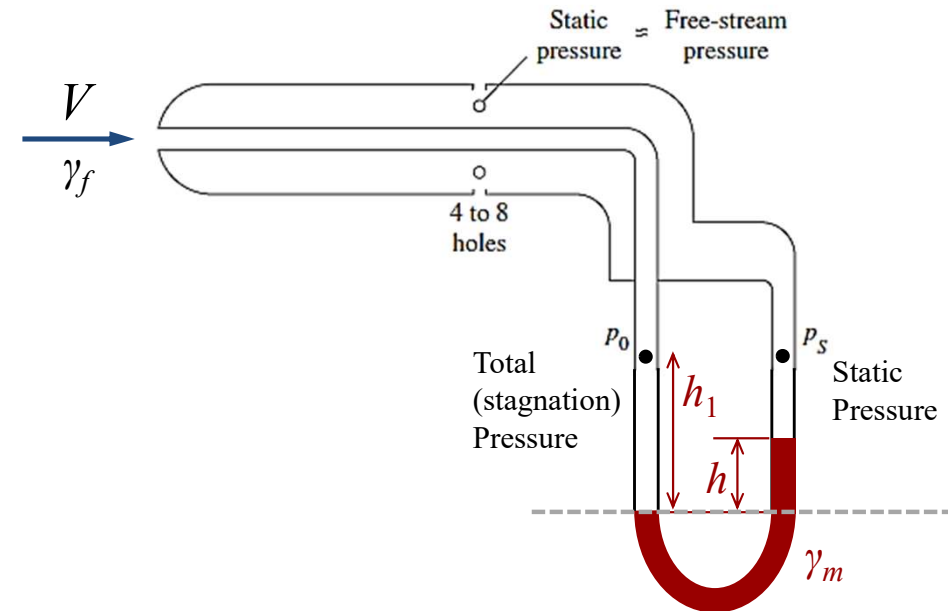
$$\Rightarrow p_0 - p_s = \gamma_f h \left(\frac{S_m}{S_f} - 1 \right)$$

So, the velocity can be measured as follows:

$$\Rightarrow V = \sqrt{\frac{2(p_0 - p_s)}{\rho_f}}$$

$$\Rightarrow V = \sqrt{2g \left(\frac{S_m}{S_f} - 1 \right) h}$$

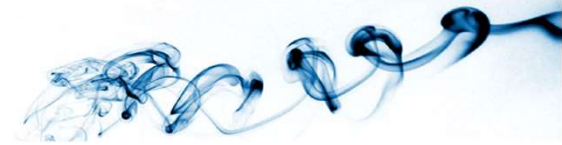
m: manometric fluid
f: flowing fluid



Pitot tube (pitot-static tube)



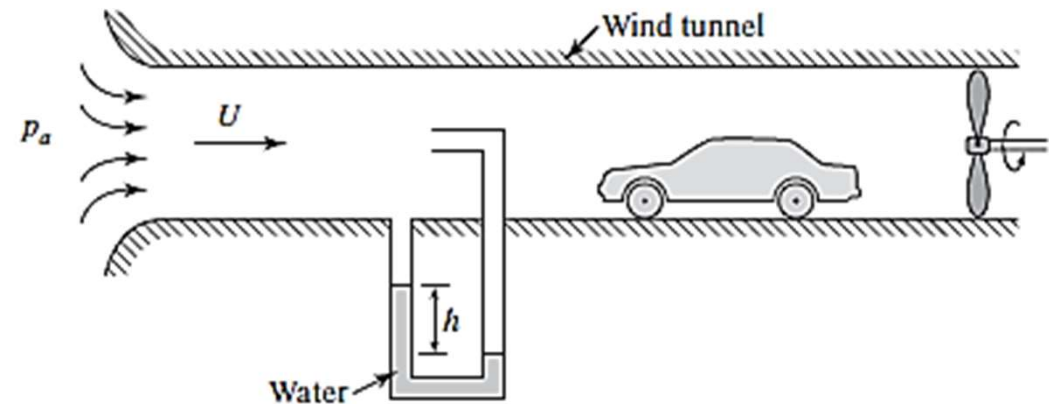
Problem



A tube is used to measure the total pressure inside a wind tunnel as shown in figure. If the water column height in the manometer $h = 8$ cm, calculate the velocity at the test section.

If the surface area of the car is 5 m^2 and measured drag force is 800 N , determine the drag coefficient at this condition.

$$\text{Hint: } C_D = \frac{F_D}{\frac{1}{2} \rho V^2 A}$$



Open Channel Flow measurement

Rectangular notch of size L x H:

Velocity at the infinitesimal element is: $v = \sqrt{2gh}$

Flowrate through the infinitesimal element is:

$$dQ_{theo.} = (Ldh)\sqrt{2gh}$$

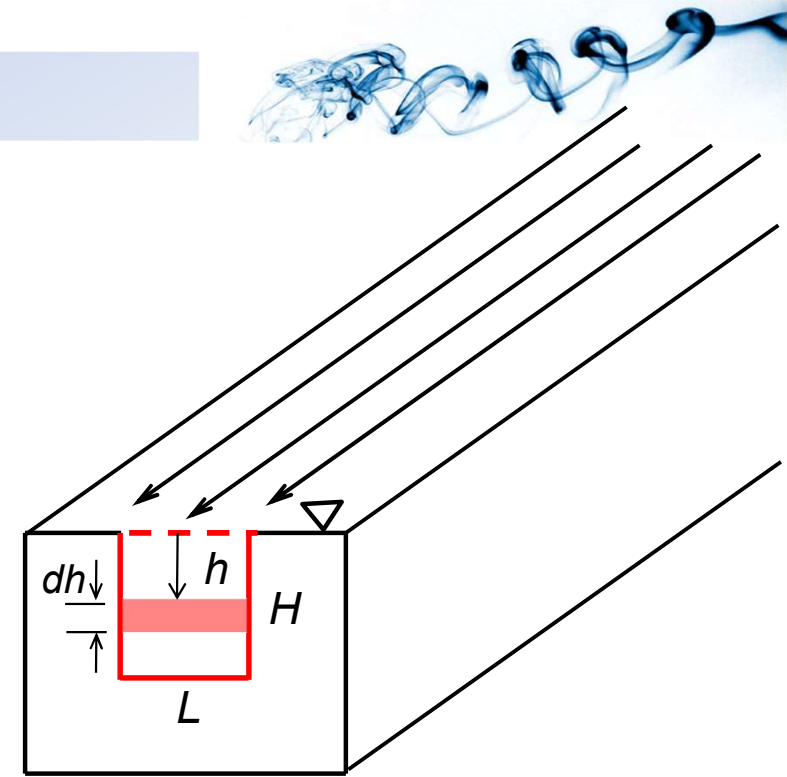
Then the actual discharge through the element becomes:

$$dQ_{actual} = C_d dQ_{theo.} = C_d (Ldh)\sqrt{2gh}$$

Then the total actual discharge :

$$\begin{aligned} Q_{actual} &= \int_0^H dQ_{actual} \\ \Rightarrow Q_{actual} &= \int_0^H C_d (Ldh)\sqrt{2gh} \\ \Rightarrow Q_{actual} &= C_d L \int_0^H \sqrt{2gh} dh \end{aligned}$$

$$\boxed{\therefore Q_{actual} = \frac{2}{3} \sqrt{2g} C_d L H^{\frac{3}{2}}}$$



Open Channel Flow measurement

Triangular notch :

$$\text{Width of the strip: } 2(H - h) \tan\left(\frac{\theta}{2}\right)$$

$$\text{Velocity at the infinitesimal element is: } v = \sqrt{2gh}$$

Flowrate through the infinitesimal element is:

$$dQ_{theo.} = \left(2(H - h) \tan\left(\frac{\theta}{2}\right) dh \right) \sqrt{2gh}$$

Then the actual discharge through the element becomes:

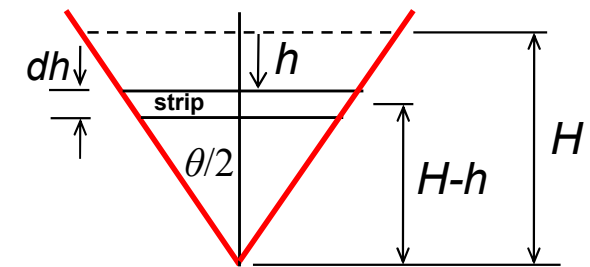
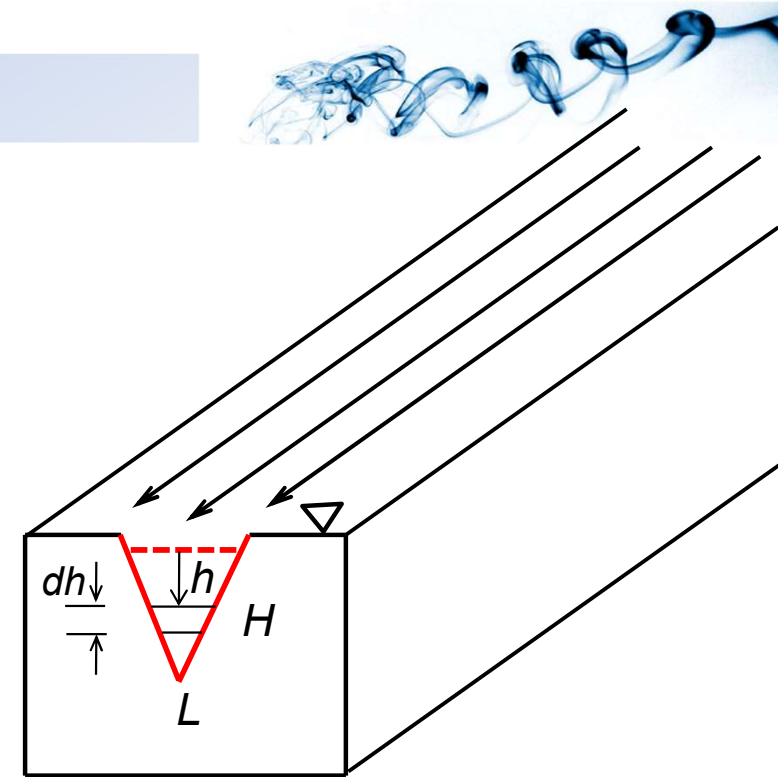
$$dQ_{actual} = C_d dQ_{theo.} = C_d 2(H - h) \tan\left(\frac{\theta}{2}\right) dh \sqrt{2gh}$$

Then the total actual discharge :

$$Q_{actual} = \int_0^H dQ_{actual}$$

$$\Rightarrow Q_{actual} = C_d \int_0^H 2(H - h) \tan\left(\frac{\theta}{2}\right) dh \sqrt{2gh}$$

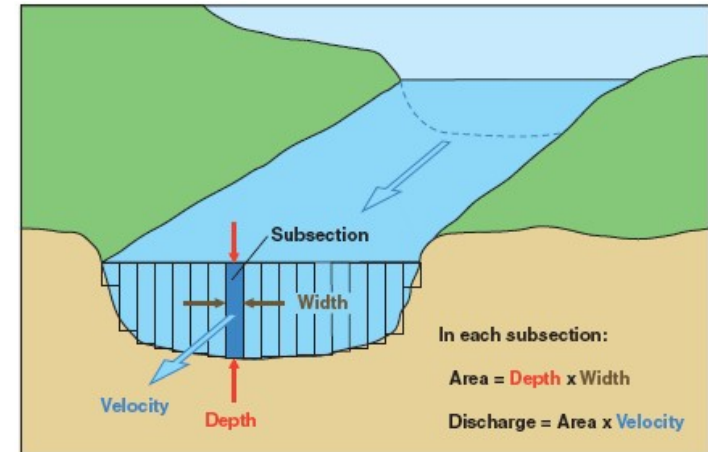
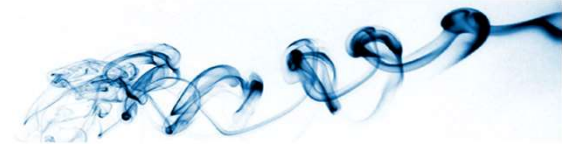
$$\therefore Q_{actual} = \frac{8}{15} \sqrt{2g} C_d \tan\left(\frac{\theta}{2}\right) H^{\frac{5}{2}}$$



Symmetric triangular notch



Current Flow meter



Current-meter discharge measurements are made by determining the discharge in each subsection of a channel cross section and summing the subsection discharges to obtain a total discharge.

